

Inside Diameter & Pressure Loss

Straight sections, Valves and Fittings

1 *Project and Fluid data:*

2	Service = Centrifugal pump discharge (water)	
3	P [bar] = Pipeline design pressure	45
4	Q [m ³ /h] = Operating flow	150
5	v [m/s] = Project Velocity	2,50
6	v [m/s] = Real Velocity in the pipeline	2,48
7	T [°C] = Operating temperature	100
8	ρ [kg/m ³] = Fluid Density	960,0
9	μ [kg/ms] = Dynamic Viscosity @ operating temperature	0,0002829
10	ν [m ² /s] = Kinematic Viscosity @ operating temperature	0,0000003

11 *Piping Data:*

12	Material = Material Metallurgy to determine pipe roughness	Commercial Steel
13	NPS [in] = Nominal Pipe Size	6
14	De [mm] = External Diameter	168,3
15	SCH [adim] = Schedule	80
16	D [mm] = Internal Diameter	146,36
17	L [m] = Straight sections length	475
18	ε [mm] = Clean pipe roughness	0,0500

19 *Total pressure loss in the system*

20	hf [mcl] = System's total pressure loss	=	26,024
21	hf [bar] = System's total pressure loss	=	2,549

22 *Pressure loss in straight sections*

23	hft [mcl] = $f^*L*v^2 / D*2g$ pressure loss in straight pipeline sections (Darcy-Weisbach)	=	16,224
24	htf [bar] =	=	1,589
25	Re [adim] = $D*\rho*v / \mu \rightarrow D*v / \nu$ Reynolds number	=	1.230.027
26	f [adim] = $64/Re$ Laminar flow $\rightarrow RE < 2000$ (Poiseuille)	=	NA
27	f [adim] = Churchill 1977, reproduces Moody in all regimes	=	0,0160
28	f [adim] = Cheng 1979, good performance for turbulent regime \rightarrow as a reference	=	0,0159

29 *Pressure loss in valves, fittings and others*

30	hff [mcl] = Pressure loss in fittings	=	9,800
31	hff [bar] =	=	0,960
32	f [adim] = Crane, commercial steel pipe, total turbulent flow \rightarrow for valves and fittings	=	0,015

N° Fittings	Angle	Ratio D/d	(f _T L/D, K)
	V obturator (m/s)		

Gate valves (K)

Full bore, threaded

0

Reduced bore, flanged

Reducing cone angle (Ø)

Ø ≤ 45°

45° < Ø ≤ 180°

27

0,75

0,000

90

0,75

0,000

Globe and angle valves, full bore and inclined (K)

Normal obturator and full bore, all joining methods

4

20,400

Inclined obturator and full bore, welded ends

0,000

90° Angle and full bore, welded or flanged ends

0,000

90° Angle and reduced, welded or flanged ends

0,75

0,000

90° Angle and full bore, threaded ends

0,000

90° Angle and reduced, threaded ends

0,75

0,000

Swing check valve, all joint types (K)

Inclined disc, threaded ends

4

6,000

Minimum velocity to lift the obturator (m/s)

0,046

Normal disc, flanged ends

0,000

Minimum velocity to lift the obturator (m/s)

0,077

Lift check valves (K)

Full bore, horizontal

Threaded ends

0,000

Reduced bore, horizontal

0,75

0,000

Minimum velocity to lift the obturator (m/s)

0,029

Reduced bore, inclined

Welded ends

0,000

Reduced bore, inclined

0,75

0,000

Minimum velocity to lift the obturator (m/s)

0,098

Tilting disc check valves, flanged (K)

Disc angle with respect to the horizontal line: 5°

50 mm ≤ Pipe Diameter ≤ 200 mm

250 mm ≤ Pipe Diameter ≤ 350 mm

400 mm ≤ Pipe Diameter ≤ 1200 mm

0,000

0,000

0,000

Minimum velocity to lift the obturator (m/s)

0,102

Disc angle with respect to the horizontal line: 15°

50 mm ≤ Pipe Diameter ≤ 200 mm

250 mm ≤ Pipe Diameter ≤ 350 mm

400 mm ≤ Pipe Diameter ≤ 1200 mm

0,000

0,000

0,000

Minimum velocity to lift the obturator (m/s)

0,041

Stop check valves, flanged (K)

Full bore, horizontal

0,000

Reduced bore, horizontal

0,75

0,000

Minimum velocity to lift the obturator (m/s)

0,040

Full bore, inclined

0,000

Reduced bore, inclined

0,75

0,000

Minimum velocity to lift the obturator (m/s)

0,055